

**A High Temperature FLiBe Loops  
Developed in Tohoku University  
in Collaboration with NIFS**

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# 1. Applicability of FLiBe as coolant

## Property of FLiBe

- Thermal characteristics :equivalent to water.
- Low electrical conductivity(low MHD resistance).
- Low reactivity with air and water.
- High melting point.
- High viscosity  $\Rightarrow$  Pressure drop by viscosity must be considered in spite of low MHD resistance.

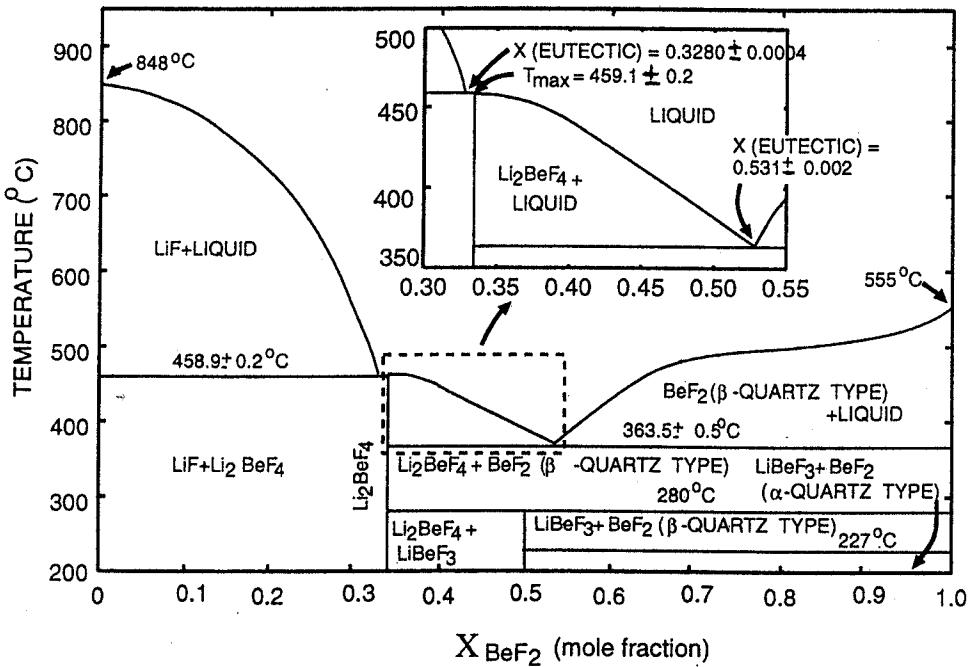


Figure 1: Revised phase diagram of the system LiF-BeF<sub>2</sub>(by K.A.Romberger, J.Braunstein, and R.E.Thoma, 1972)

## Comparison between FLiBe and Li

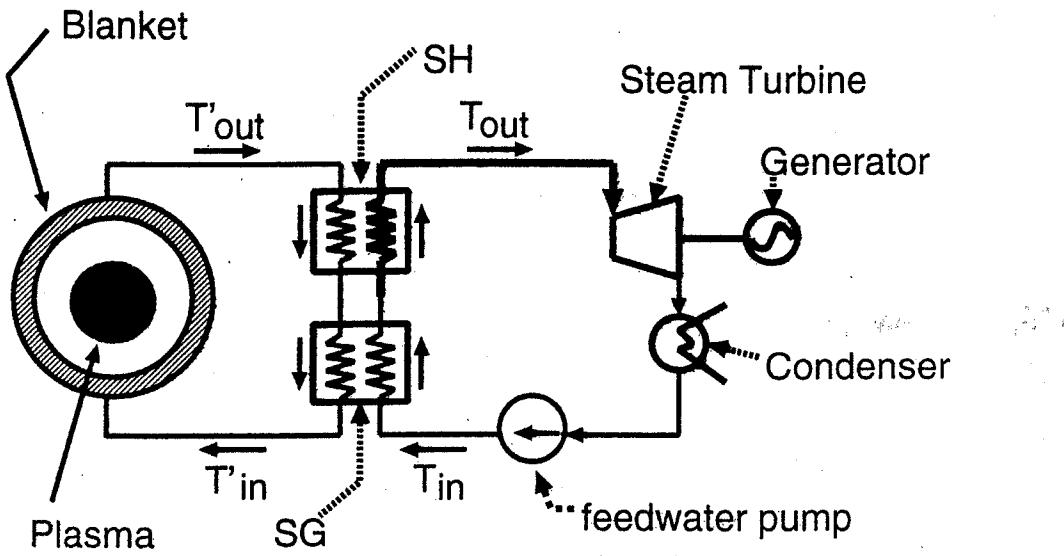


Figure 2: Concept of heat exchange

◊ Assumptions ◊

→ Fusion energy:  $3000 \text{ MW}_{th}$

→  $3000 \text{ MW}_{th}$  is exchanged at HX. And . . .

About 80 % of total fusion energy is received in blanket.

Minimum temperature :  $450^\circ\text{C}$  (melting point and viscosity)

Maximum temperature :  $650^\circ\text{C}$  (corrosion on structure materials)

→ FLiBe:  $2.5(\text{m}^3/\text{s})$  ; Li:  $2.9(\text{m}^3/\text{s})$

⇒ Calculation of pressure drop by viscosity and

MHD(steady flow in a pipe).

## 2.Calculation of pressure drop

Consider fully developed steady-state flow:

### Pressure drop by viscosity

Using Darcy-Weisbach Equation:

$$\frac{dP}{dl} = \lambda \frac{1}{D} \frac{\rho U_0^2}{2} \quad (Pa/m) \quad (1)$$

where, pipe friction coefficient  $\lambda = 64/Re$ (laminar flow)

$$\lambda = 0.184/Re^{0.2} \text{ (turbulent flow)}$$

### Pressure drop by MHD

$$\frac{dP}{dl} = \frac{C}{1+C} \sigma_f U_0 B_0^2 \sin^2 \alpha \quad (Pa/m) \quad (2)$$

where,  $\alpha$ :angle between fluid flow and line of B.

$$C \simeq \frac{\sigma_w(b-a)}{\sigma_f a} : \text{wall-to-fluid conductance ratio.}$$



Pressure drop by MHD varies with  $\alpha$   
and insulating rate of pipe.

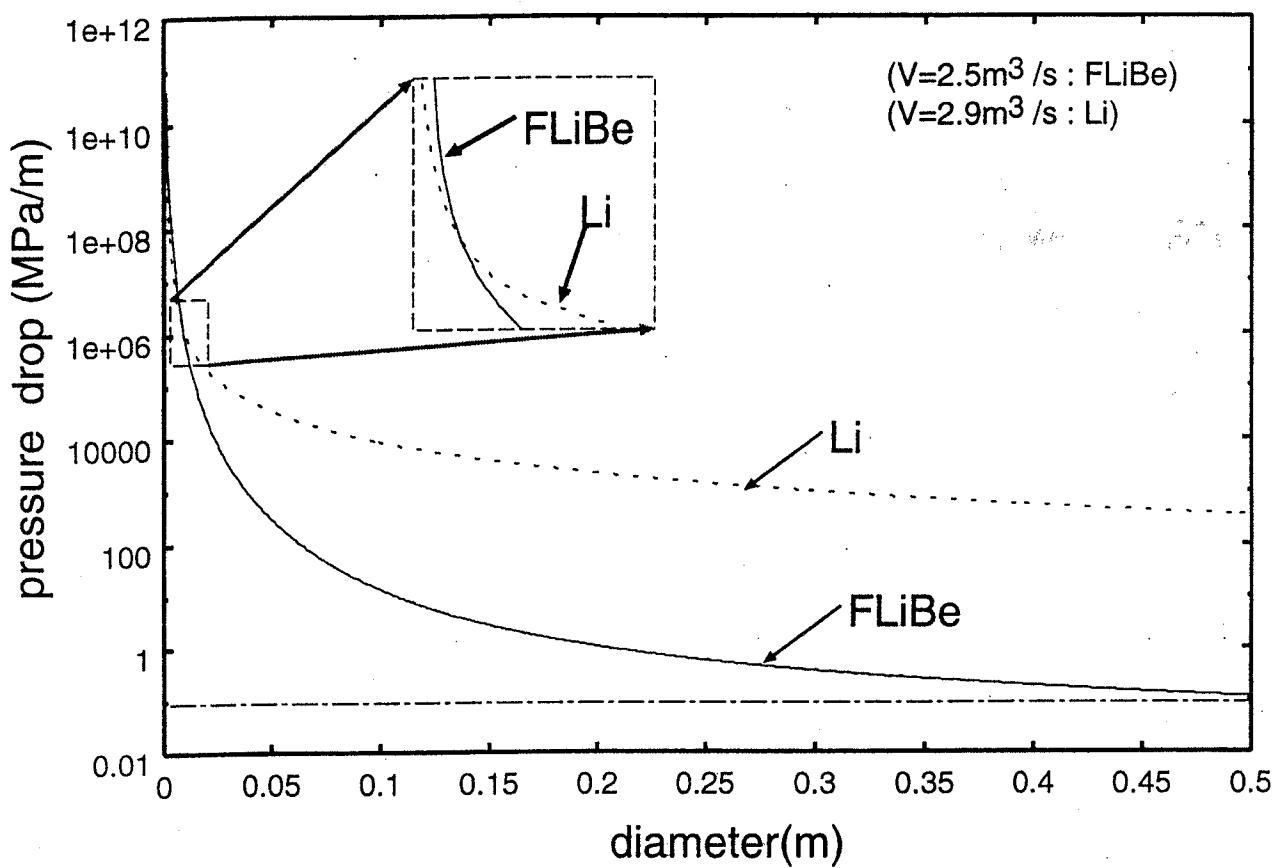


Figure 3: Pressure drop of FLiBe and Li, by viscosity and MHD ( $B^*=12\text{Tesla}$  ;  $\alpha^{**}=90^\circ$  )

\* Magnetic Field

\*\*angle between Fluid Flow and Magnetic Field

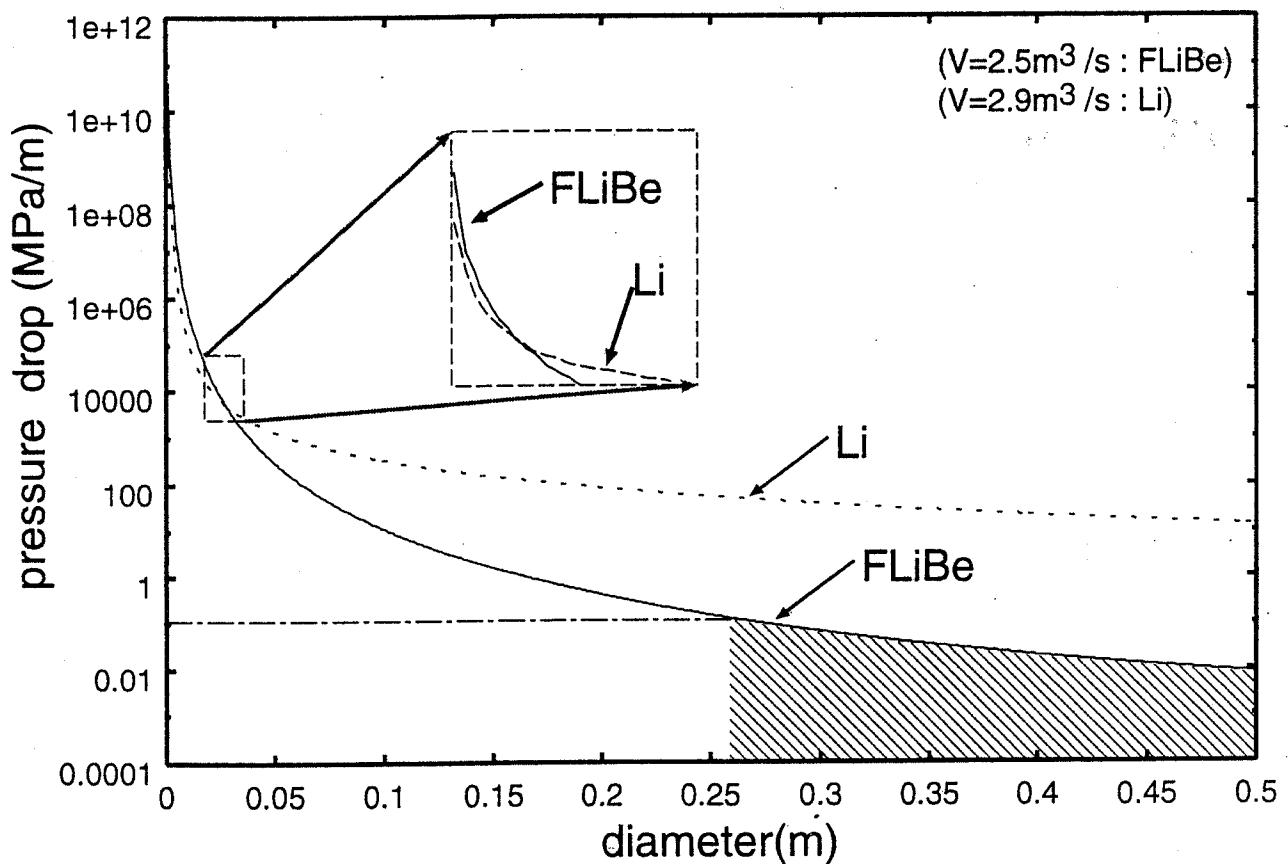


Figure 4: Pressure drop of FLiBe and Li, by viscosity and MHD ( $B=12\text{Tesla}$  ;  $\alpha=10^\circ$  )

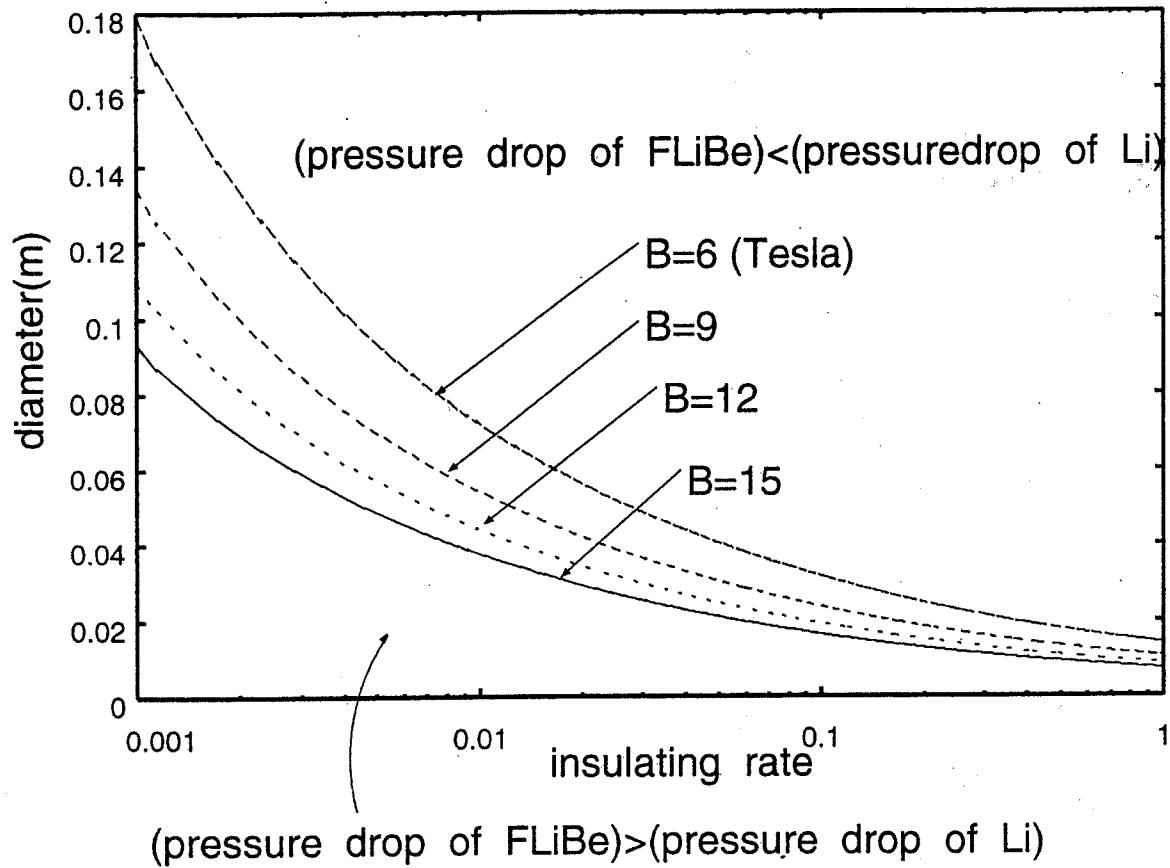


Figure 5: Equivalent pipe diameter from view point of pressure drop by viscosity and MHD ( $\alpha=90^\circ$ )

### 3. FLiBe loop

#### 3. 1 Purpose

To develop the FLiBe blanket system,  
we need

- 1) some experience on FLiBe loop  
for both thermohydraulics and  
safe treatment of FLiBe
- 2) new technique of heat removal in  
divertor
- 3) date on compatibility with material  
and other chemical characteristics.

## 3.2 Research Plan

1997 FY

- specification of FLiBe loop  
(temp., flow rate, material)
- designing the loop system

1998 FY

- final designing of the loop  
(Apr.-Aug.)
- fabrication(Aug.-Feb.)  
→ almost finished
- settlement(Feb.)  
→ delayed until this March

Cleaning the experimental facility  
Regulation about Be

- flow experiment(May or June)
- fundamental study on surface heat removal
- fabrication of test section for heat removal (pre experiment)

1999 FY

- development of high heat removal system
- planing chemical experiments
- designing & fabrication of the test section

2000 FY

- experiments on heat removal, control & chemical characteristics

### 3.3 FLiBe Loop

#### 1) Configurations

Location : Tohoku University, Japan

Size : 2m(L) x 2.5m(D) x 3m(H)

Loop type : closed loop with forced air cooling

Test section : removable, 70 cm length

Flow rate : 10 l/min

temp. range: 773K - 873 K

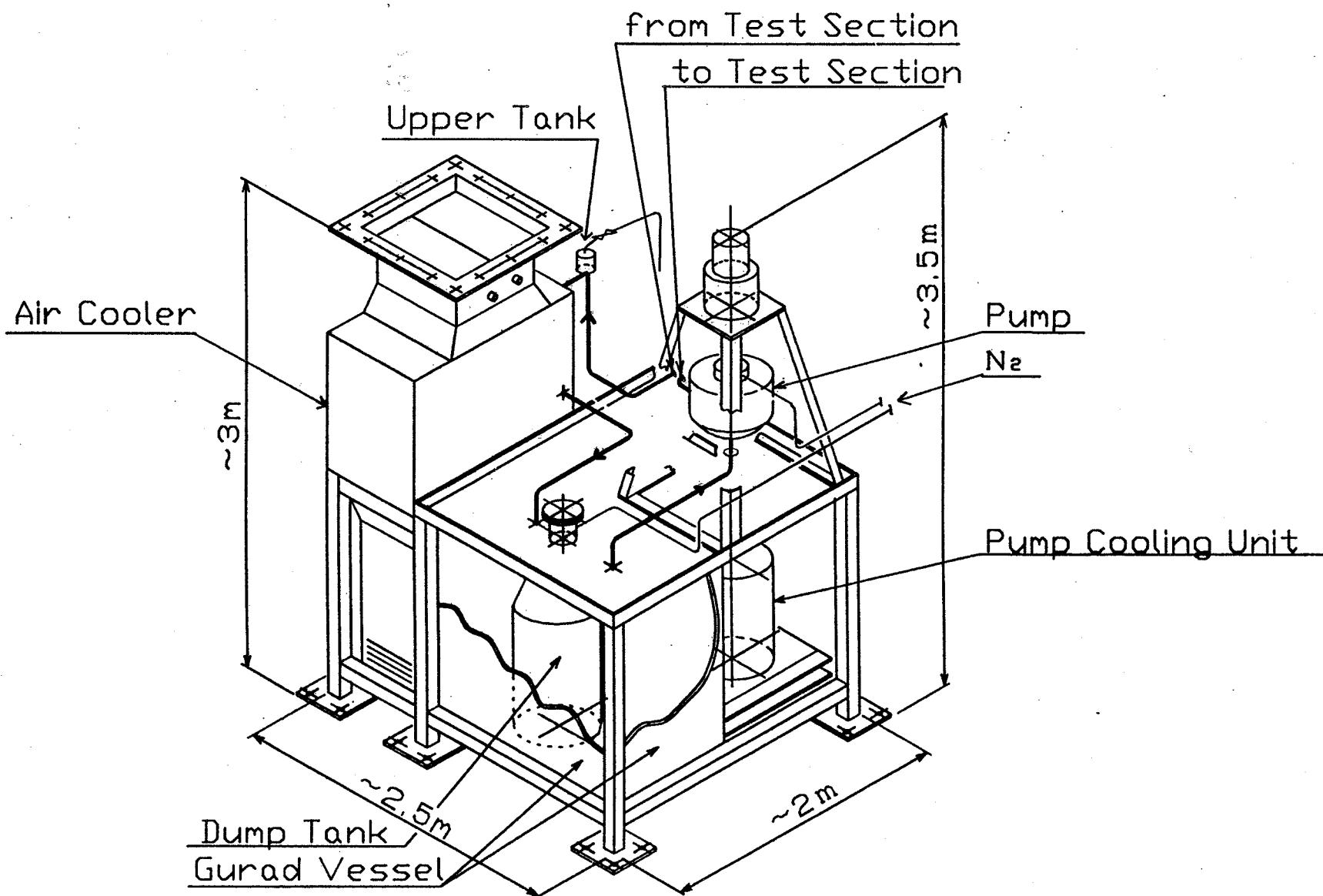
Heat transfer : 80 kW

Max. press. : 0.7 MPa

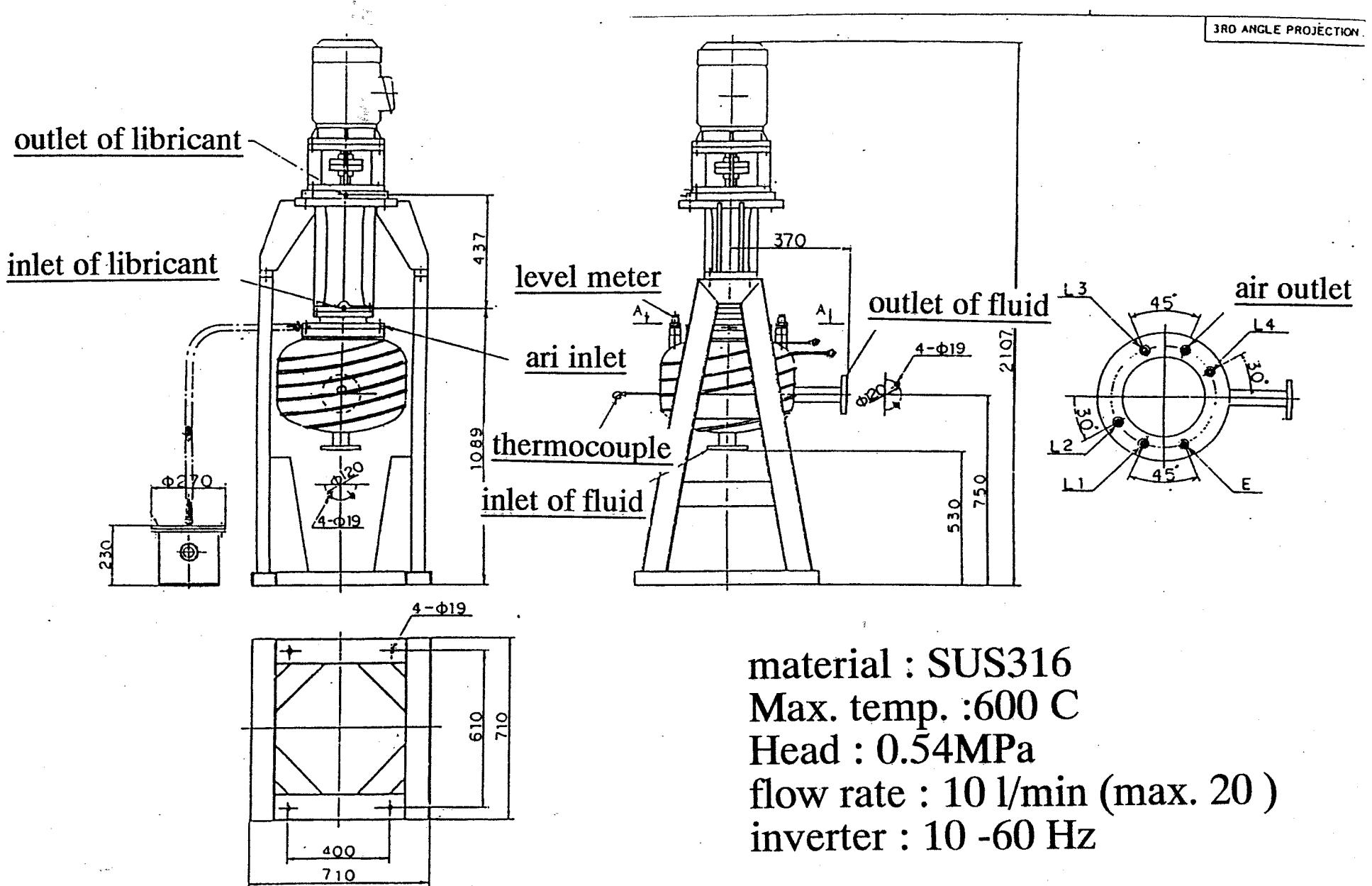
Allowable press. drop at test section :  
0.4MPa

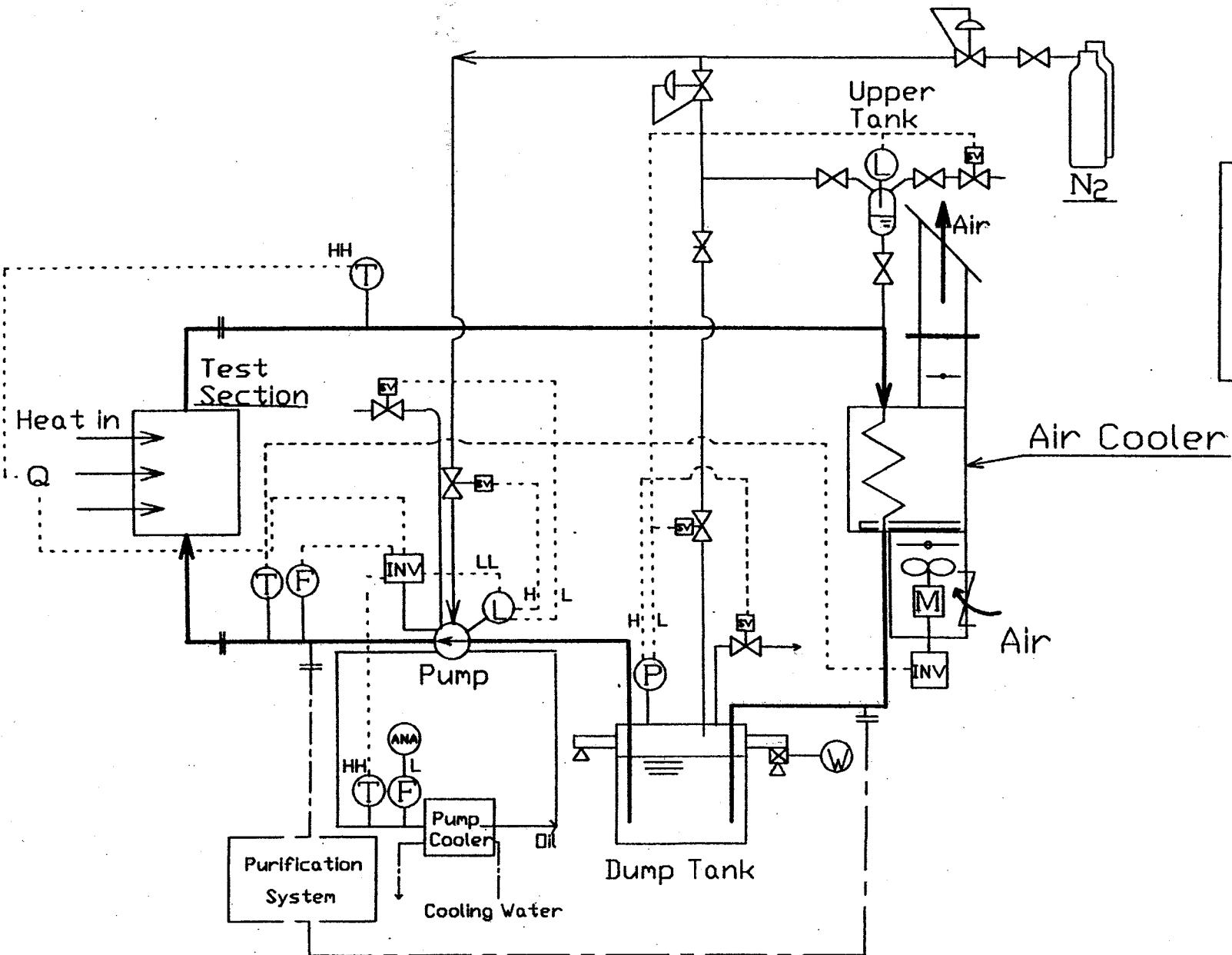
Str. Material : SUS316

#### 2) Schematic view of the loop



## Molten Salt(FLIBE) Circulating Equipment





<u>Specification</u>	
Fluid	: FLIBE
Max. Flow	: 10L/min
Design Temp.	: 650 °C
Design Press.	: 0.7 MPa
Material	: SUS316

<u>Symbol</u>	
(P)	: Press.
(T)	: Temp.
(F)	: Flow
(L)	: Level
(W)	: Load Cell

Molten Salt(FLIBE) Circulating System

## 4. Improvement of thermal characteristics

High Pr Number

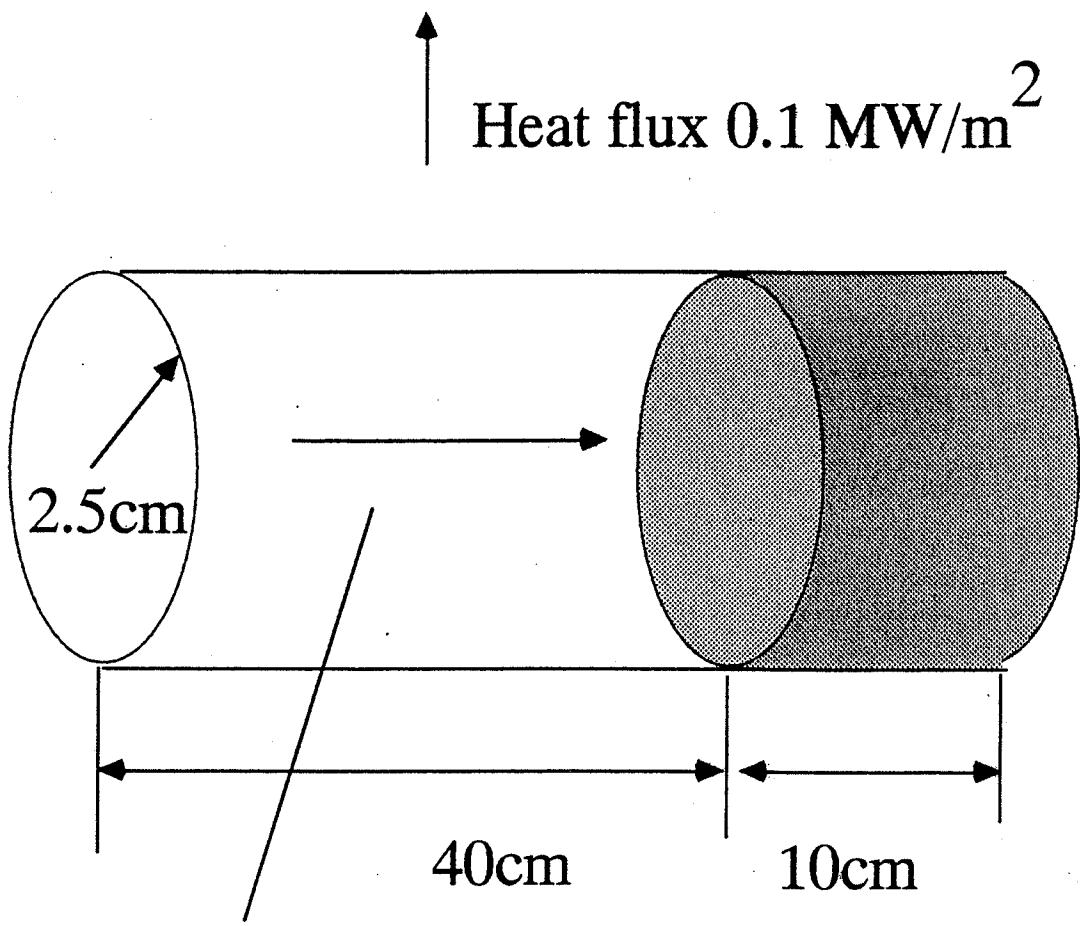


Thin thermal boundary layer

Enhancement for heat transfer

○Swirl tube ( Feb.17)

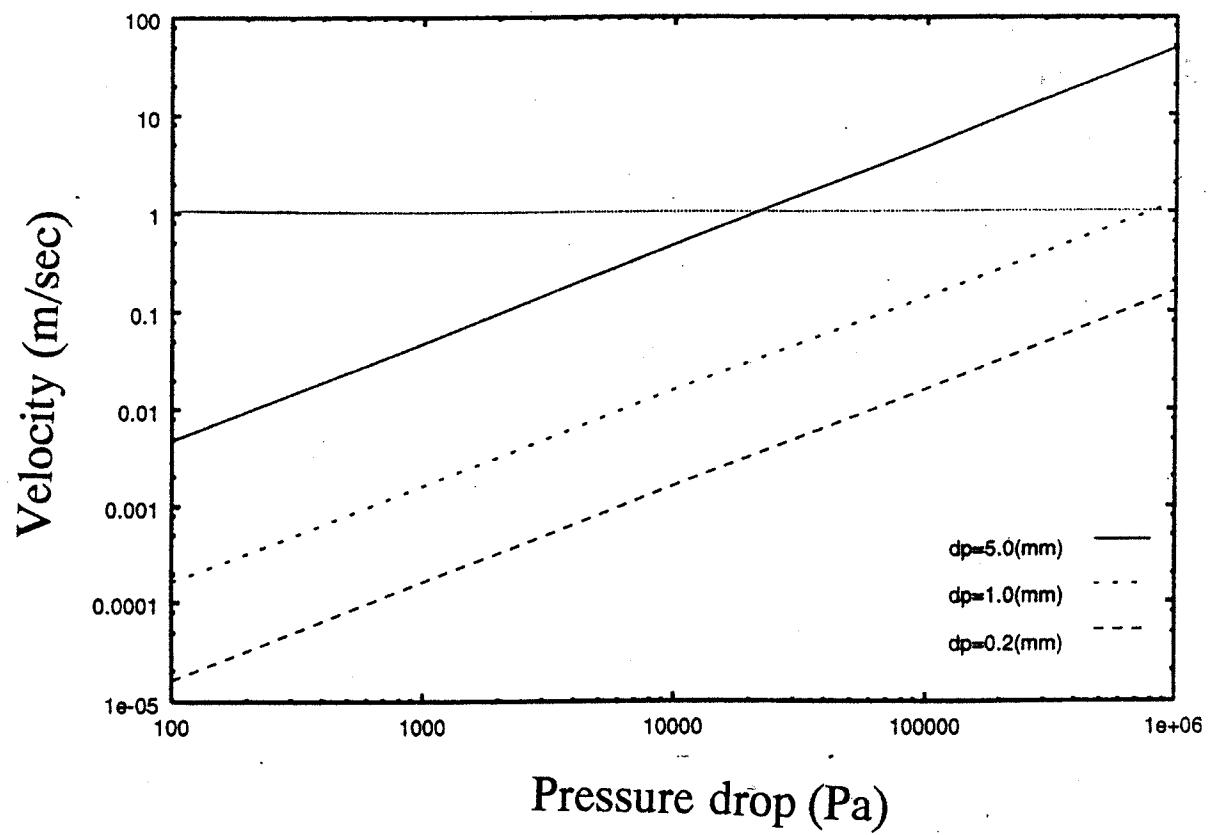
○Porous media (Feb.16)

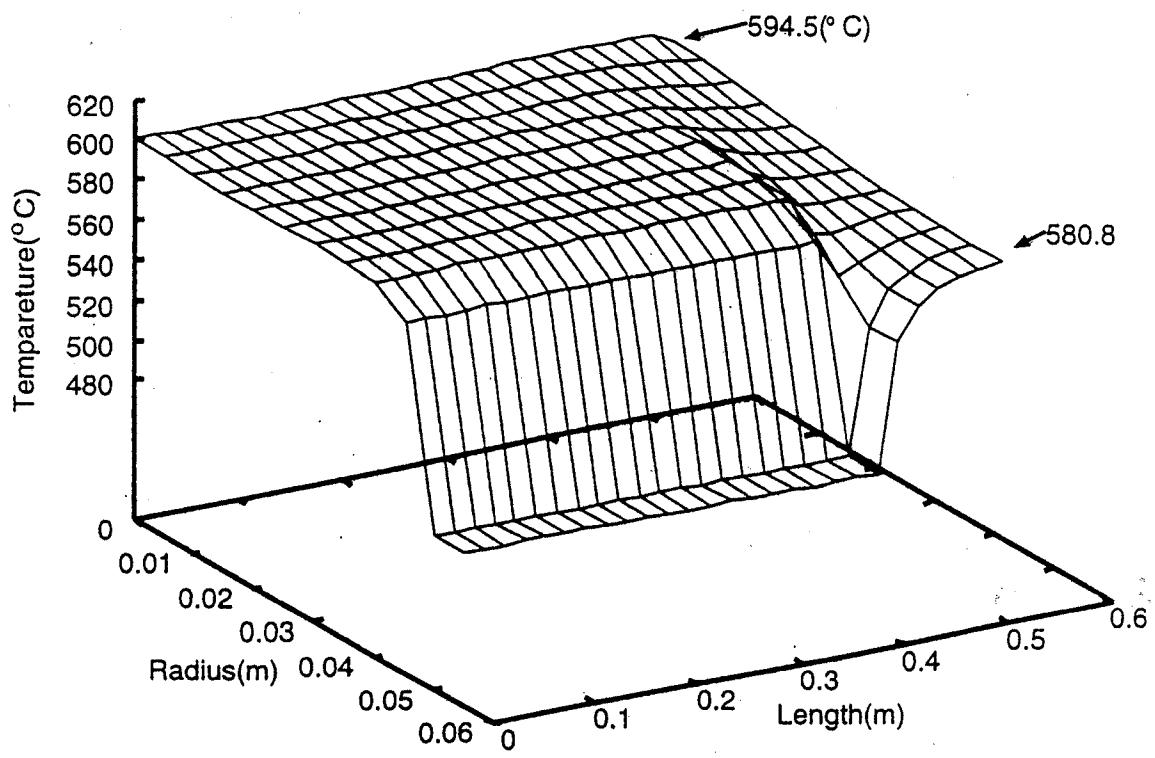


Fully developed flow

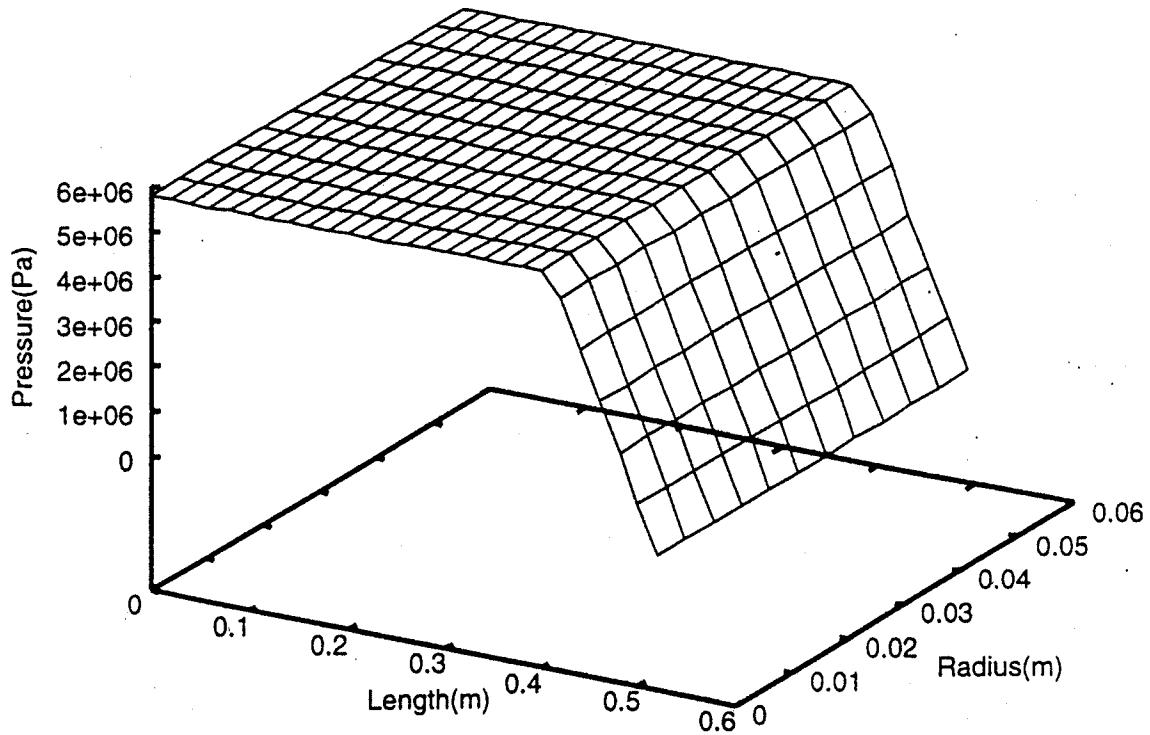
porosity 0.4

A model used in numerical analysis





Temperature distribution (1 m/sec,  $d=0.2\text{mm}$ )



Pressure distribution (1 m/sec,  $d=0.2\text{mm}$ )